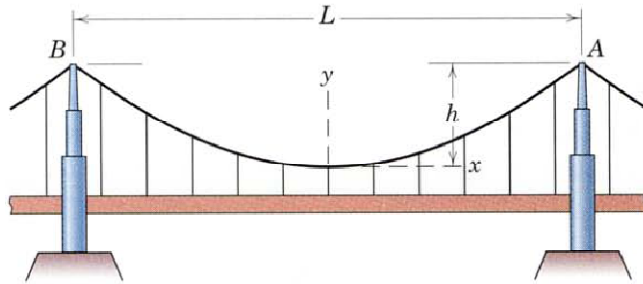


5.8 Flexible Cables

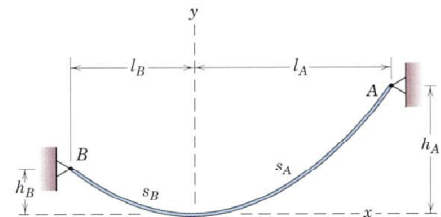
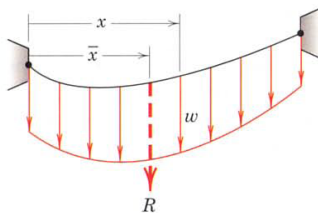
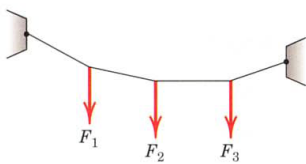
J. L. Meriam and L. G. Kraige, Engineering Mechanics, Statics 5th ed., SI

- Examples: suspension bridges, transmission lines, messenger cables for supporting heavy trolley or telephone lines.

- To determine for design purposes :
Tension force (T),
span (L), sag (h),
length of the cable (s).



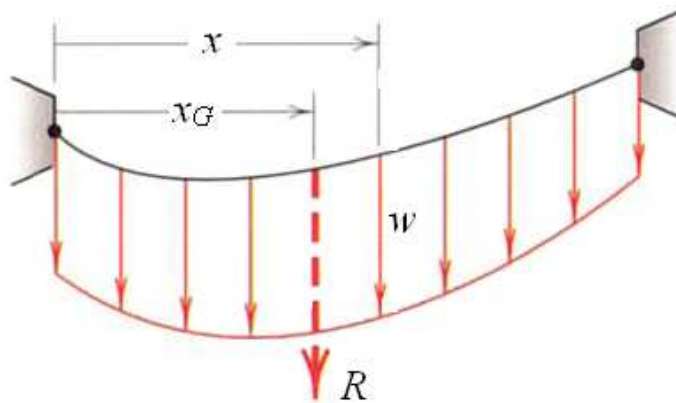
- Assume: any resistance offered to bending is negligible.
means: the tension force in the cable is *always* in the direction of the cable.
- Flexible cables may support
 - concentrated loads.
 - distributed loads
 - its own weight only
 - all three or only two of the above



- In several cases the weight of the cable may be negligible compared with the loads it supports.

General Relationships

- Assume:
 - the distributed load w (in N/m) is *homogeneous* and has a *constant* thickness.
 - distributed load $w = w(x)$.



- The resultant \mathbf{R} of the vertical loading $w(x)$ is

$$R = \int dR = \int w dx \quad (1)$$

- Position of \mathbf{R}

$$x_G = \frac{\int x dR}{R} \quad (2)$$

x_G center of gravity, which equals the centroid of the area if w is homogeneous.

Static Equilibrium

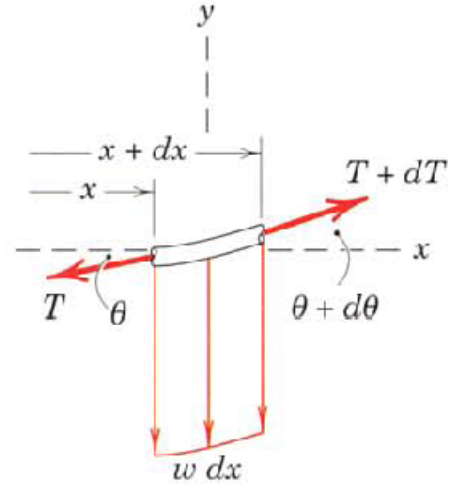
Note that the changes in both T and θ are taken to be positive with a positive change in x

$$\uparrow : \sum F_y = 0$$

$$(T + dT) \sin(\theta + d\theta) = T \sin \theta + w dx \quad (3)$$

$$\rightarrow : \sum F_x = 0$$

$$(T + dT) \cos(\theta + d\theta) = T \cos \theta \quad (4)$$



With the equalities:

$$\sin(a \pm b) = \sin a \cos b \pm \cos a \sin b$$

$$\cos(a \pm b) = \cos a \cos b \mp \sin a \sin b$$

and the substitutions $\sin(d\theta) = d\theta$, $\cos(d\theta) = 1$, which hold in the limit as $d\theta$ approaches zero, yields

$$(T + dT)(\sin \theta + \cos \theta d\theta) = T \sin \theta + w dx \quad (5)$$

$$(T + dT)(\cos \theta - \sin \theta d\theta) = T \cos \theta \quad (6)$$

Neglecting the second-order term ($dT d\theta$) and simplifying leads to

$$T \cos \theta d\theta + dT \sin \theta = w dx \quad (7)$$

$$-T \sin \theta d\theta + dT \cos \theta = 0 \quad (8)$$

which can be written in the form

$$d(T \sin \theta) = w dx \quad (9)$$

$$d(T \cos \theta) = 0 \quad (10)$$

Equation (10) means that the horizontal component of T remains constant.

$$T \cos \theta = T_H = \text{const.} \quad (11)$$

$$\rightarrow T = \frac{T_H}{\cos \theta} \quad (12)$$

Substituting Eq. (12) into Eq. (9) yields

$$d(T_H \tan \theta) = w dx \quad (13)$$

with $\tan \theta = \frac{dy}{dx}$, Eq. (13) becomes

$$\frac{d^2 y}{dx^2} = \frac{w}{T_H} \quad (14)$$

which represents the differential equation for the flexible cable.

The solution of this equation with considering the boundary conditions yields the shape of the cable $y = y(x)$.

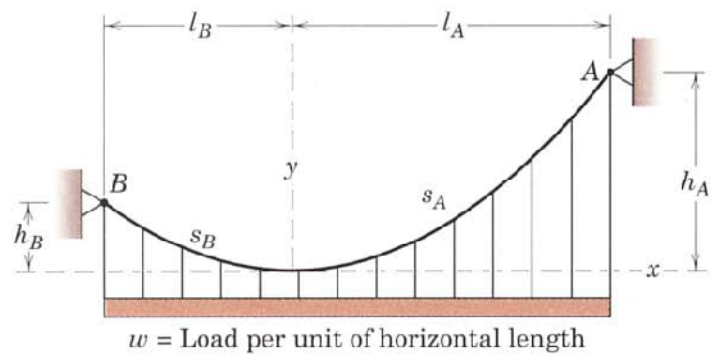
Parabolic Cable

Assume: $w = \text{const.}$, load homogeneous

Example: suspension bridge

mass of the cable \ll mass of the bridge \rightarrow neglect the cable mass

Note: The mass of the cable itself is not distributed uniformly with the horizontal (x -axis).



- Place the coordinate origin at the lowest point of the cable.

$$\frac{d^2y}{dx^2} = \frac{w}{T_H} \quad (14)$$

Integrating yields

$$\frac{dy}{dx} = \frac{wx}{T_H} + C_1 \quad (15)$$

$$y(x) = \frac{wx^2}{2T_H} + C_1x + C_2 \quad (16)$$

Boundary conditions:

$$\text{a) } x = 0, \quad \frac{dy}{dx} = 0 \quad \text{Eq. (15)} \rightarrow C_1 = 0$$

$$\text{b) } x = 0, \quad y = 0 \quad \text{Eq. (16)} \rightarrow C_2 = 0$$

$$\rightarrow y(x) = \frac{wx^2}{2T_H} \quad (17)$$

Horizontal tension force T_H

At the lowest point, the tension force is horizontal.

BC: at $x = l_A, y = h_A$

Substituting this boundary condition into Eq. (17) gives

$$h_A = \frac{wl_A^2}{2T_H} \quad (18)$$

$$\rightarrow T_H = \frac{wl_A^2}{2h_A} \quad (19)$$

Note that T_H is the minimum tension force in the cable ($T_H = T_{min}$).

Tension force $T(x)$

From the figure we get

$$T = \sqrt{T_H^2 + w^2 x^2} \quad (20)$$

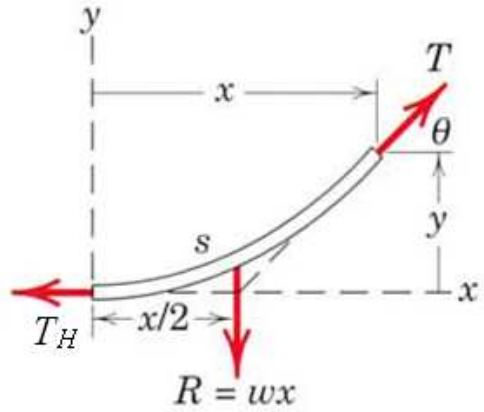
Where T becomes maximum for $x = x_{max}$, since T_H and w are constants.

Using Eq. (19) yields

$$T = w\sqrt{x^2 + (l_A^2 + 2h_A)^2} \quad (21)$$

The maximum tension force occurs at $x = x_{max}$; in this case $x_{max} = l_A$.

$$T_{max,A} = wl_A\sqrt{1 + (l_A/2h_A)^2} \quad (22)$$



The length of cable (s)

Length s_A

Integrating the differential length

$$ds = \sqrt{(dx)^2 + (dy)^2} \quad (23)$$

gives

$$\int_0^{s_A} ds = \int_0^{l_A} \sqrt{1 + (dy/dx)^2} dx \quad (24)$$

a) exact solution

$$s_A = \frac{1}{2a} \left[x\sqrt{x^2 + a^2} + a^2 \ln (x + \sqrt{x^2 + a^2}) \right]_0^{l_A}$$
$$s_A = \frac{1}{2a} \left[l_A \sqrt{l_A^2 + a^2} + a^2 \ln (l_A + \sqrt{l_A^2 + a^2}) - a^2 \ln a \right] \quad (25)$$

where

$$a = \frac{T_H}{w} = \frac{l_A^2}{2h_A} \quad (26)$$

b) approximate solution

using the binomial series

$$(1 + x)^n = 1 + nx + \frac{n(n-1)}{2!} x^2 + \frac{n(n-1)(n-2)}{3!} x^3 + \dots \quad (27)$$

which converges for $x^2 < 1$, and replacing x by $(wx/T_H)^2$ and setting $n = 1/2$, we get

$$s_A = l_A \left[1 + \frac{2}{3} \left(\frac{h_A}{l_A} \right)^2 - \frac{2}{5} \left(\frac{h_A}{l_A} \right)^4 + \dots \right] \quad (28)$$

This series is convergent for values of $h_A/l_A < 1/2$, which holds for most practical cases.

For the cable section from the origin to B (x rotated 180°), we obtain in a similar manner by replacing h_A , l_A and s_A by h_B , l_B and s_B , respectively

$$T_H = \frac{wl_B^2}{2h_B} \quad (29)$$

$$T = w\sqrt{x^2 + (l_B^2 + 2h_B)^2} \quad (30)$$

$$T_{max,B} = wl_B\sqrt{1 + (l_B/2h_B)^2} \quad (31)$$

$$s_B = \frac{1}{2a} \left[l_B\sqrt{l_B^2 + a^2} + a^2 \ln \left(l_B + \sqrt{l_B^2 + a^2} \right) - a^2 \ln a \right] \quad (32)$$

where in this case

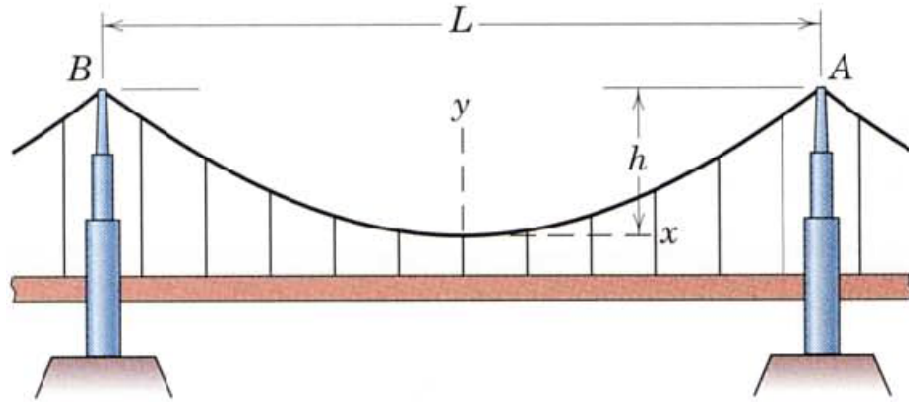
$$a = \frac{l_B^2}{2h_B} \quad (33)$$

Approximate solution

$$s_B = l_B \left[1 + \frac{2}{3} \left(\frac{h_B}{l_B} \right)^2 - \frac{2}{5} \left(\frac{h_B}{l_B} \right)^4 + \dots \right] \quad (34)$$

Since $h_A > h_B$, the absolute maximum tension force in the cable will naturally occur at end A , since this side of the cable supports the greater proportion of the load.

Symmetric case



$$s_A = s_B, \quad l_A = l_B, \quad h_A = h_B$$

total span $L = 2l_A$, total sag $h = h_A$

In this case we get

$$T_{max} = \frac{wL}{2} \sqrt{1 + (L/4h)^2} \quad (35)$$

$$s = 2h \left[\sqrt{1 + b^2} + b^2 \ln \left(\frac{L}{2} + \frac{L}{2} \sqrt{1 + b^2} \right) - b^2 \ln \left(\frac{L^2}{8h} \right) \right] \quad (36)$$

where

$$b = \frac{L}{4h} \quad (37)$$

Approximate solution:

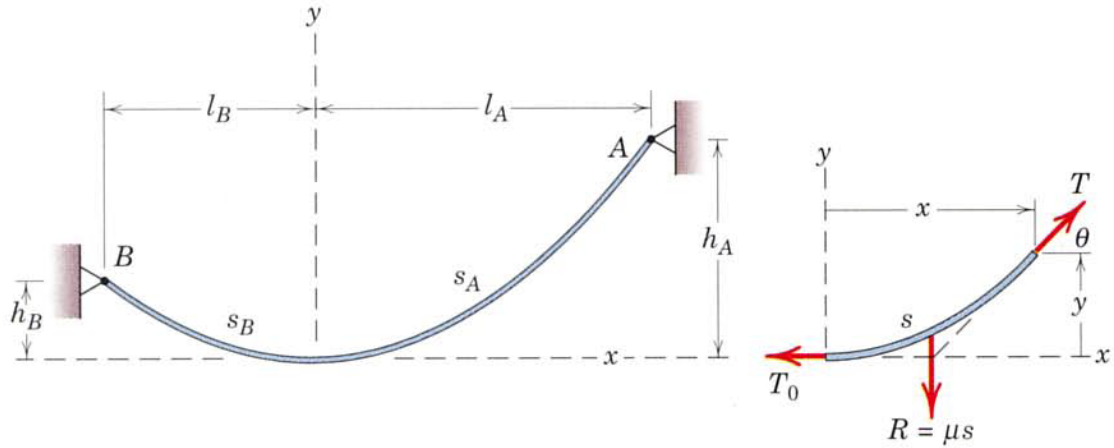
$$s = L \left[1 + \frac{8}{3} \left(\frac{h}{L} \right)^2 - \frac{32}{5} \left(\frac{h}{L} \right)^4 + \dots \right] \quad (38)$$

This series converges for all values of $h/L < 1/4$. In most cases $h \ll L/4$.

→ The first three terms of series (38) give a sufficiently accurate approximation.

Catenary Cable

Consider cable weight only



$$wx \rightarrow \mu s; \quad wdx \rightarrow \mu ds$$

where μ is the weight per unit length of the cable in N/m.

$$\text{Eq. (10): } T_H = T \cos \theta \rightarrow T = \frac{T_H}{\cos \theta}$$

Substituting into Eq. (9) and replacing $w dx$ with μds yields

$$\begin{aligned} d(T_H \tan \theta) &= \mu ds, & \tan \theta &= \frac{dy}{dx} \\ \rightarrow d\left(T_H \frac{dy}{dx}\right) &= \mu ds \end{aligned} \quad (39)$$

Differentiation with respect to x yield

$$\frac{d^2 y}{dx^2} = \frac{\mu}{T_H} \frac{ds}{dx} \quad (40)$$

With $ds = \sqrt{(dx)^2 + (dy)^2}$, we get

$$\frac{d^2 y}{dx^2} = \frac{\mu}{T_H} \sqrt{1 + \left(\frac{dy}{dx}\right)^2} \quad (41)$$

Substitution: $p = \frac{dy}{dx} \quad \rightarrow \quad \frac{dp}{dx} = \frac{d^2y}{dx^2}$ (42)

$$\rightarrow \quad \frac{dp}{\sqrt{1+p^2}} = \frac{dx}{c} \quad (43)$$

where $c = \frac{T_H}{\mu}$

Substituting $p = \sinh u$, $dp = \cosh u du$, $u = \operatorname{arsinh} p$, gives

$$\frac{dx}{c} = \frac{\cosh u}{\sqrt{1+\sinh^2 u}} du \quad (44)$$

Integrating leads to

$$\frac{x}{c} = u + C_1 \quad (45)$$

Boundary condition:

At $x = 0$, $\frac{dy}{dx} = p = 0 \rightarrow \sinh(0) = 0 \rightarrow C_1 = 0$

So that

$$u = \frac{x}{c} = \operatorname{arsinh} p \quad (46)$$

or

$$p = \frac{dy}{dx} = \sinh \frac{x}{c} \quad (47)$$

which leads to

$$dy = \sinh \frac{x}{c} dx \quad (48)$$

Integrating yields

$$y = c \cosh \frac{x}{c} + C_2 \quad (49)$$

Boundary condition:

$$\text{At } x = 0, y = 0 \quad \rightarrow \quad C_2 = -c$$

Thus, we obtain the equation of the curve formed by the cable

$$y = \frac{T_H}{\mu} \left(\cosh \frac{\mu x}{T_H} - 1 \right) \quad (50)$$

Cable length

From the free-body diagram shown in the figure we see that

$$\frac{dy}{dx} = \tan \theta = \frac{\mu s}{T_H} \quad \rightarrow \quad s = \frac{T_H}{\mu} \frac{dy}{dx}$$

Using Eq. (47), we get then

$$s = \frac{T_H}{\mu} \sinh \frac{\mu x}{T_H} \quad (51)$$

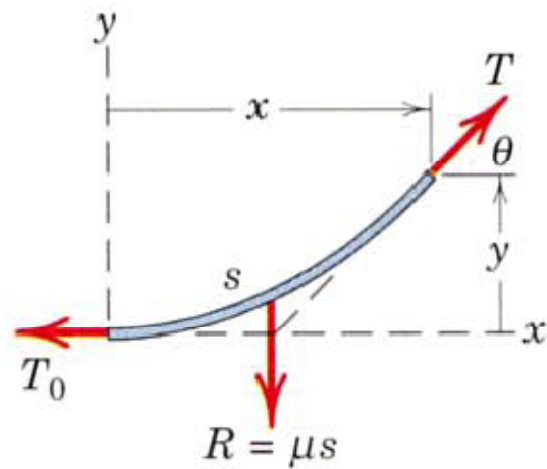
Where the unknown minimum tension force T_H may be obtained from Eq. (50) by using the boundary condition $y = h_A$ at $x = l_A$.

Tension force

From the figure, we get

$$T^2 = \mu^2 s^2 + T_H^2 \quad (52)$$

Substituting Eq. (51) into Eq. (52) leads to



$$T^2 = T_H^2 \left(1 + \sinh^2 \frac{\mu x}{T_H} \right) = T_H^2 \cosh^2 \frac{\mu x}{T_H} \quad (53)$$

or

$$T = T_H \cosh \frac{\mu x}{T_H} \quad (54)$$

With equation (50) we get

$$T = T_H + \mu y \quad (55)$$

The solution of catenary problems where the sag-to-span ratio is small may be approximated by the relations developed for the parabolic cable. A small sag-to-span ratio means a tight cable, and the uniform distribution of weight along the cable is not very different from the same load intensity distributed uniformly along the horizontal.